Tidal Turbine Benchmarking Project: Stage II – Head Waves

Oxford: Richard Willden, Xiaosheng Chen, Nijmeh Marouf, Yadong Han, Sam Tucker Harvey, Ian Campbell, Chris Vogel, Federico Zilic de Arcos

Bath: Anna Young, Ian Benson, Elias Marchetti

Edinburgh: Ross Calvert

Hull: Jim Gilbert, Kaushal Bhavsar, Tom Allsop

Manchester: Tim Stallard, Hannah Mullings

31st October 2025

This project is being funded jointly by The UK EPSRC Supergen ORE Hub EP/S000747/1 & EP/Y016297/1, RHJW's EPSRC Fellowship EP/R007322/1 & the EPSRC CoTide programme EP/X03903X/1.

















Benchmarking Project: Overview and Objectives

Unsteady loading and the inability to confidently predict unsteady loading and to quantify errors drives unnecessary redundancy and design conservatism.

Objectives:

- i. improve accuracy of modelling techniques,
- ii. improve confidence in the use of modelling techniques,
- iii. quantify modelling errors for different techniques under different loading scenarios,
- iv. development of novel measurement techniques.

Approach:

- conduct a large laboratory test of a highly instrumented tidal turbine in waves and turbulent currents to provide underlying data,
- ii. conduct a series of community wide (academia and industry) blind prediction exercises with staged data release, leading to open access datasets.



















Requirements, Tests & Facility

Test requirements:

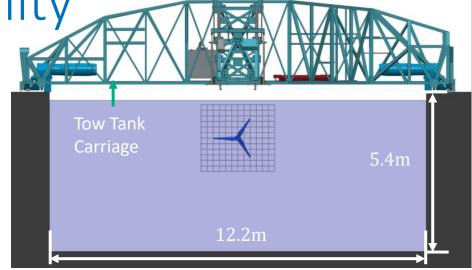
- Require low blockage experiments with a large diameter rotor for in-blade sensing and Reynolds independence,
- Flume options blockage too high,
- Tow tank low blockage but turbulence low,
- Solution: tow tank with an upstream turbulence grid

Test conditions:

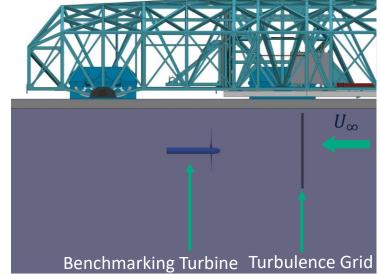
- Stage 1: Uniform flow
 Uniform flow + Grid generated turbulence
- Stage 2a: Uniform flow + Head Waves
 Yawed uniform flow
- Stage 2b: Uniform flow + Following Waves

QinetiQ towing tank facility, Haslar, Portsmouth UK

- 270m (L) x 12.2m (W) x 5.4m (D)
- Tow speed 1m/s
- Tow length approx. 150m, settling time ~15mins.























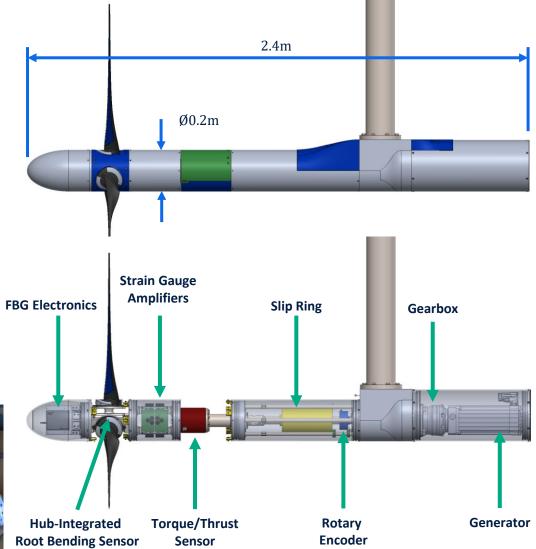


Instrumented Turbine

- 1.6m diameter rotor / 0.2m diameter nacelle
- Two blades instrumented with strain gauges at six radial locations for flapwise and edgewise bending moments
- Remaining blade instrumented with fibre Bragg sensors
- Individual root blade moments measured with hub – integrated root bending sensors
- Torque and Thrust measured by shaft mounted transducer upstream of front bearing















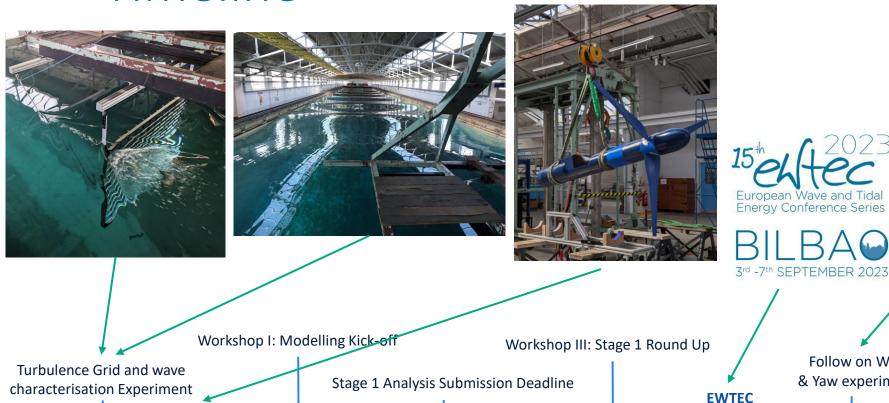




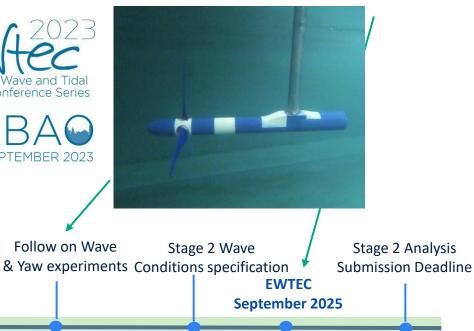




Timeline









June 2022

April 2022

Workshop II: Stage 1
Data Presentation

2nd November 2022

Workshop IV: Stage 1 – prediction review. Stage 2 – Release of first set of Wave Conditions

March 2025

Workshop V: Stage 2
Data Discussion



July 2021





5th September 2022



17th January 2023



September 2023





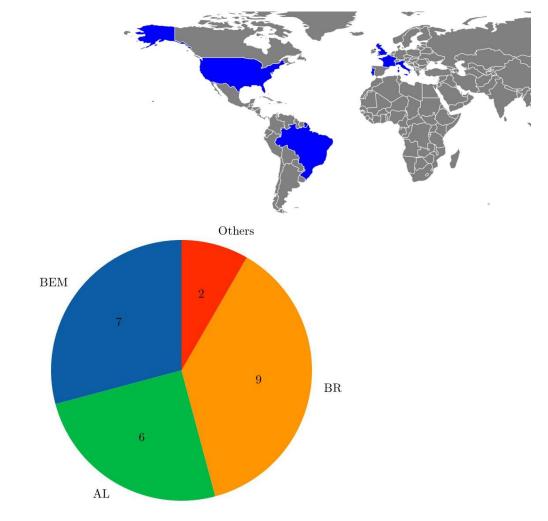
15th April 2025



16th January 2026

Stage 1: Benchmarking Participants

- 12 collaborating research groups:
 - from across academia and industry
 - from 6 countries; UK, France, Italy, Portugal Brazil & USA.
- 26 submissions from a wide range of methods falling into 5 categories:
 - Blade Element Momentum (BEM)
 - Blade Resolved CFD (BR)
 - Actuator Line CFD (AL)
 - Boundary Integral Equation Model (BIEM)
 - Vortex methods



















Benchmarking cases

	Low Turbulence (LT) Cases												
Case No.	I	II	III	IV	V	VI	VII	VIII	IX	X	XI	XII	XIII
\mathbf{U}_{∞} [m/s]	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0	1.0
λ	4.02	4.52	5.03	5.36	5.53	5.78	6.03	6.53	6.70	7.04	7.20	7.54	7.87

	Elevated Turbulence (ET) Cases											
Case No.	I	II	III	IV	V	VI	VII	VIII	IX	X	XI	
\mathbf{U}_{∞} [m/s]	0.9207	0.9207	0.9207	0.9207	0.9207	0.9207	0.9207	0.9207	0.9207	0.9207	0.9207	
λ	3.91	4.46	4.91	5.37	5.64	5.82	6.19	6.37	6.92	7.37	7.73	

- Participants asked to concentrate on priority (yellow) cases.
- LT cases submitted by 24 participants, ET cases submitted by 18 participants.

















Participants: Blade Resolved

Submission	Solver	Turbulence Model	State	Wall Treatment	Mesh Size	Flow Domain	Free Surface	
blueOASIS-	ReFRESCO	k-ω SST	Steady	Resolving	34.3M	MRF *	None	
BR-RANS	2.8.0					WR §		
CHE-BR-	STAR-CCM+	k-ω SST	Unsteady	Resolving	3.5M	TTG†	VOF ‡	
uRANS	STAR-CCM+	κ-ω 331	Unsteady	Resolving	5.5WI	110	VOF #	
CNR-INM-	X NAVIS	SA	Unsteady	Resolving	24M	MRF WR	None	
BR-uRANS	(in-house)	SA	Unsteady	Resolving	24IVI	WIKF WK	None	
LOMC-BR-	OmenEOAM	1 COT	Ctoody	Dagalyina	26M	MDE	None	
RANS	OpenFOAM	$k - \omega$ SST	Steady	Resolving	26M	MRF	140110	
NREL-BR-	STAR-CCM+	$k - \omega$ SST	Stoody	Function	4.23M	MRF	None	
RANS	STAR-CCM+	$\kappa - \omega$ 331	Steady	runction	4.25111	WIKF	None	
NREL-BR-	STAR-CCM+	$k - \omega$ SST	Unsteady	Function	21.15M	TTG	VOF	
uRANS	STAR-CCM+	$\kappa - \omega$ 331	Offsteady	runction	21.1511	110	VOI	
UoE-BR-	OpenFOAM	$k - \omega$ SST	Steady	Resolving	18.8M	MRF	None	
RANS	OpenFOAM	$\kappa - \omega$ 331	Steady	Resolving	10.01/1	WIKF	None	
UoO-BR-	On an EOAD #	1 CCT	Ct a a day	Danaladas	2014	MDE	Nama	
RANS	OpenFOAM	$k - \omega$ SST	Steady	Resolving	38M	MRF	None	
USP-BR-DES	OpenFOAM	DES SST	Unsteady	Function	33.6M	TTG	None	

^{*} MRF = multiple reference frame technique with by default a 120° cylindrical wedge domain of a single blade, or if specified the § WR = whole rotor geometry. † TTG = tow-tank geometry with rotating turbine submerged at experimental depth. ‡ VOF = Volume-of-Fluid free surface representation.

















Participants: Actuator Line

Submission	Solver	Turbulence Model	Mesh Size	2D Polars	Polar Interpolation	Free Surface Rep.	Nacelle Model	Tip-loss Model	
QUB-AL-	0 5047	LES	16004	Provided	0: 1 1	MOE 8	N.T.	G) f :	
LES	OpenFOAM	Smagorinsky	16.3M	*	Single polar	VOF §	None	SM ‡	
UoM-AL-	STREAM	1 CCT	0.0414	D	0:11-	NIDEC 6	ID ±	NI	
uRANS	(in-house)	k-ω SST	0.94M	Provided	Single polar	NDFS §	IB†	None	
UoM-AL-	DOFAS	I EO WALE	2014	Duari da d	Cinala mala :	NIDEC	TD	CM	
LES	(in-house)	LES WALE	30M	Provided	Single polar	NDFS	IB	SM	
UoO-AL-	OmanEOAM	L COT	C 0114	Duarridad	Tu	NDEC	D 1 1	G) 4	
NRSM	OpenFOAM	k-ω SST	6.91M	Provided	Interpolation	NDFS	Resolved	SM	
UoO-AL-	OnenEOAM	l CCT	6 01M	Dwaridad	Tu	NDEC	Dagalyad	337N#	
NRWM	OpenFOAM	k-ω SST	6.91M	Provided	Interpolated	NDFS	Resolved	WM	
UoO-AL-	On an EOAM	L COT	10.007		Tu	NDEC	TD.	WD 6 !	
IBWM	OpenFOAM	k - ω SST	10.80M	Provided	Interpolated	NDFS	IB	WM ‡	

^{*2}D performance polars provided as part of the benchmarking exercise (from 2D RANS). § VOF = Volume of Fluid deform-able free surface, NDFS = Non-Deformable Free Surface. † IB = Immersed Boundary method. ‡ SM & WM = Shen et al.- and Wimshurst & Willden-type tip-loss models.

















Participants: Blade Element Momentum

Submission	2D Polars	Polar Interpolation	Turbulent Inflow	Induction/Wake Correction	Root Model	Tip-loss Correction	
LOMC-BEM	2D RANS	Re-Tu Interpolation	None	Modified Turbulent Wake Model	None	PDL†	
NREL-BEM	Provided * with Rotation Correction	Single polar	None	Buhl Model	PDL	PDL	
SU-BEM	Provided	Single polar	Sandia Method	High-induction Model	PDL	PDL	
UoE-BEM	XFOIL	Re Interpolation	Spectral Method	Buhl Model	GLT ‡	GLT	
UFU-BEM- Aerodas	Provided with Aerodas Correction	Single polar	None	None	PDL	PDL	
UFU-BEM- SD	Provided with Stall Delay Correction	Single polar	None	None	PDL	PDL	
UoM-BEM-1	Provided	Single polar	None	GLT	None	GLT	
UoM-BEM-2	Provided Single polar		Import from LES	GLT	GLT	GLT	

^{* 2}D performance polars provided as part of the benchmarking exercise (from 2D RANS). † PDL = Prandtl-type hub / tip correction.

















[‡] GLT = Glauert-type induction / hub / tip correction.

Other Participants:

Submission	Method Type	Description			
		Boundary Integral Equation Model. Uses a Viscous Flow Correction model to			
CND INM DIEM D12 / D22	DIEM	estimate the effects of viscosity on blade loads with input 2D flow lift and drag			
CNR-INM-BIEM-D12 / D22	BIEM	Boundary Integral Equation Model. Uses a Viscous Flow Correction model to estimate the effects of viscosity on blade loads with input 2D flow lift and drag curves calculated using XFOIL. D12 / D22 refers to alternative shape parameters used in modelling the curvature of the wake surface in the tip vortex region. 3D unsteady Lagrangian Vortex Particle Method. Blades represented by lifting			
		used in modelling the curvature of the wake surface in the tip vortex region.			
I OMC Venter	DD.	3D unsteady Lagrangian Vortex Particle Method. Blades represented by lifting			
LOMC-Vortex	BR	line model using tabulated 2D lift and drag coefficients.			









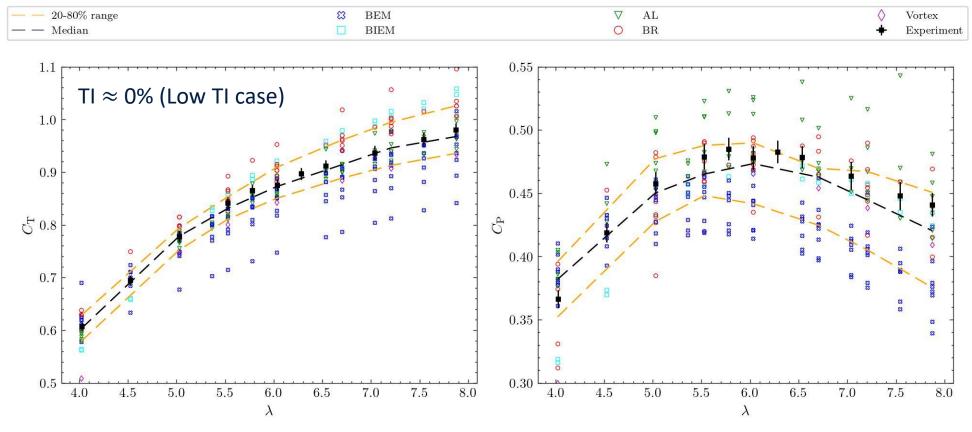








Blind prediction results



- Power and thrust coefficients are generally well predicted. 20-80% prediction interval particularly good,
- 20-80% Thrust predictions are more tightly banded ($\pm 5\%$) than Power ($+7\% \rightarrow -11\%$),
- AL, BEM, BR, BIEM, Vortex, exhibit different biases, with results spread often linked to choice of sub-models.









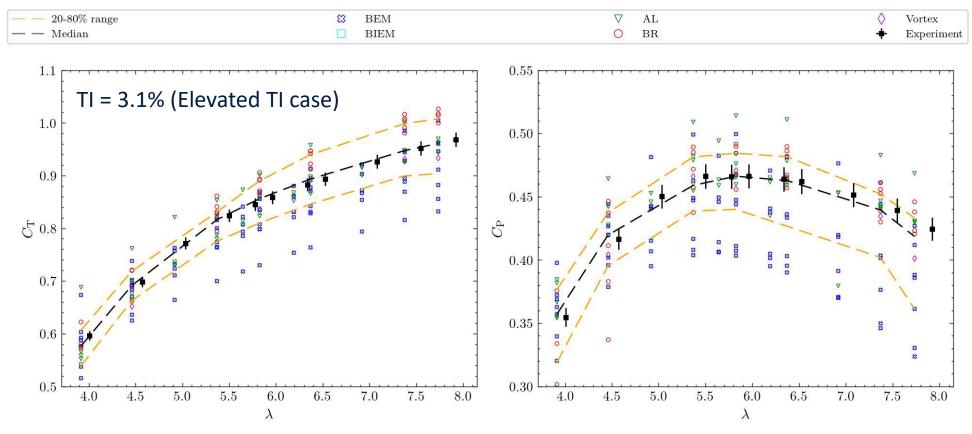








Blind prediction results



- BR tendency to underpredict C_P and overpredict C_T .
- BEM methods tend to underpredict both.
- BIEM over-predicts C_T but C_P good at high TSR.

- Vortex method consistently under-predicts LT cases, but more accurate for ET cases.
- AL methods good alignment in both C_P and C_T .

















Reduction in Prediction Uncertainty

Definition of data submission levels

- Level 1 (L1) completely blind submission
- Level 2 (L2) "user-error corrected" submissions correction for data input, setup errors etc
- Level 3 (L3) New results that use improved modelling techniques / approaches building on data comparisons from this and other exercises.

Improvements from **L1** to **L2** result from having a reliable dataset against which to verify model setup.

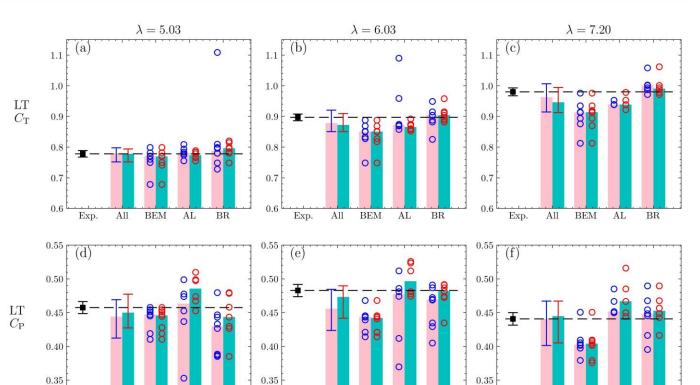
Table: Standard deviations of L1 and L2 solutions.

	Low Turbulence (LT) Cases										
		C_{1}	Γ	C_P							
	All	BEM	AL	BR	All	BEM	AL	BR			
L1	11.8	6.02	17.1	17.3	15.49	5.15	10.5	15.6			
L2	5.45	5.86	2.58	4.80	6.93	4.96	1.64	3.88			

	Elevated Turbulence (ET) Cases									
	C_T C_P									
	All	BEM	AL	BR	All	BEM	AL	BR		
L1	14.7	7.71	2.26	22.8	16.5	6.55	1.54	22.3		
L2	6.22	7.44	5.04	3.03	7.87	6.15	4.33	2.56		

Figure: Medians and ranges of C_T and C_P for fully blind (L1) and user-error-corrected (L2) submissions, TI ~ 0% (LT case)

















Exp.





Reduction in Prediction Uncertainty

This has already **provided quantifiable improved confidence in simulation model application**.

Standard deviations of solutions reduced by over 50% from c. 15% at L1 to 7% at L2 for All cases (methods, TSRs, TIs, C_T and C_P).

Further improvements to accuracy (**L3**) being sought by modellers through improvements and refinements to modelling techniques using benchmarking data as reference data set.

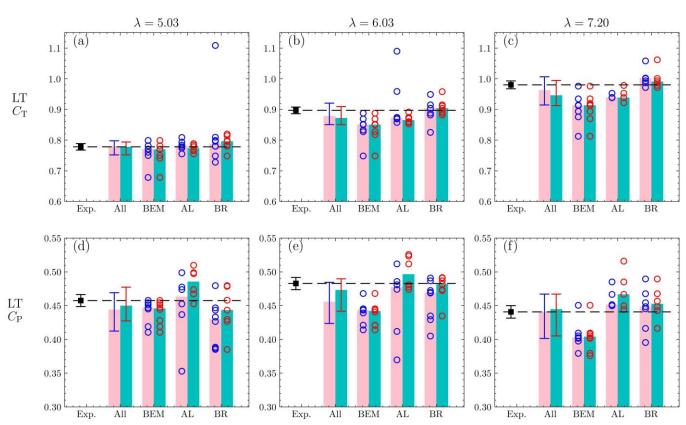
Table: Standard deviations of L1 and L2 solutions.

	Low Turbulence (LT) Cases										
		C_{2}	Γ		C_P						
	All	BEM	AL	BR	All	BEM	AL	BR			
L1	11.8	6.02	17.1	17.3	15.49	5.15	10.5	15.6			
L2	5.45	5.86	2.58	4.80	6.93	4.96	1.64	3.88			

	Elevated Turbulence (ET) Cases									
	C_T C_P									
	All	BEM	AL	BR	All	BEM	AL	BR		
L1	14.7	7.71	2.26	22.8	16.5	6.55	1.54	22.3		
L2	6.22	7.44	5.04	3.03	7.87	6.15	4.33	2.56		

Figure: Medians and ranges of C_T and C_P for fully blind (L1) and user-error-corrected (L2) submissions, TI ~ 0% (LT case)





















Bending Moments – Experiments & CFD

- Experimental data for spanwise distributed Flapwise (FW) and Edgewise (EW) BMs enables

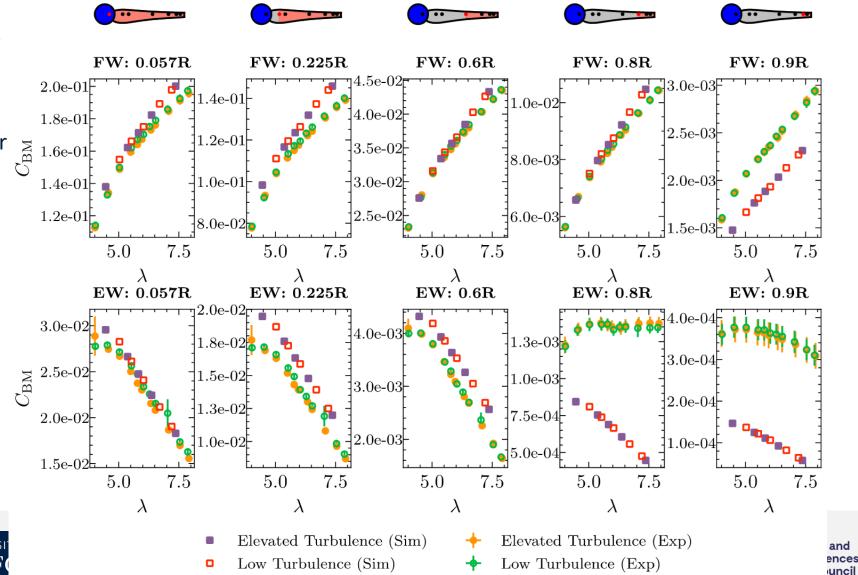
 assessment of model

 performance at a more granular level.

 1.6e-01
- FW and EW bending moment coefficients

$$C_{BM} = M_{BM} \frac{16}{\rho \pi D^3 U_{\infty}^2}$$

- Minor changes between Low and Elevated TI levels,
- CFD simulations over-predict EW in mid-span locations & under-predict FW & EW at tip,
- CFD Root BM well predicted.



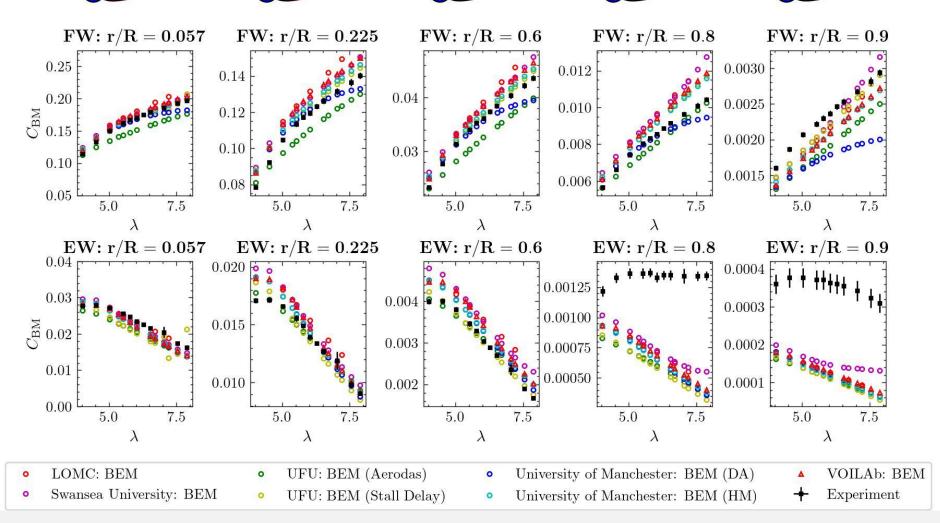






Bending Moments – BEM & BIEM

- BEM models tend to under-predict inboard bending moments, overpredict through midspan up to 0.8R (FW) and 0.6R (EW), and then underpredict further outboard.
- Divergence in model predictions outboard due to choice of tip correction & high thrust turbulent wake model.
- Over/under predictions lead to net underprediction in $C_T \& C_P$.













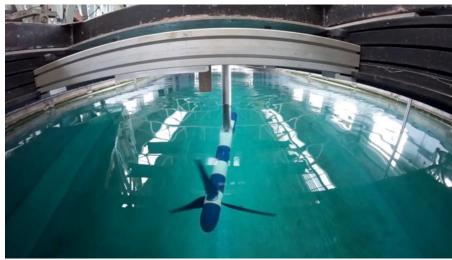




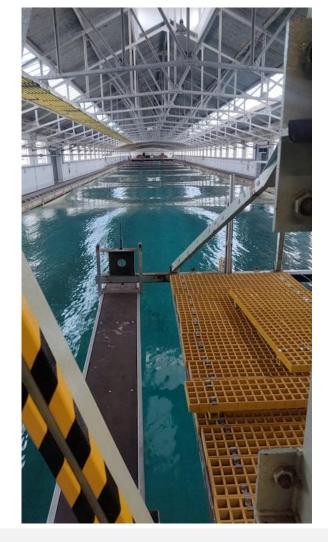


Stage 2: Experiments in waves

- Turbine tested at QinetiQ, March 3rd-21st & 6-24th October 2025
- Wave characterization using 3 different techniques – 7 solid gauges, 6 ultrasonic probes and a rake of "barnacle" 5-hole probes
- Wave experiments covering >50
 wave conditions
- Additional steady flow experiments with yawed turbine
- Total of > 400 tests performed























Selection of Wave Conditions

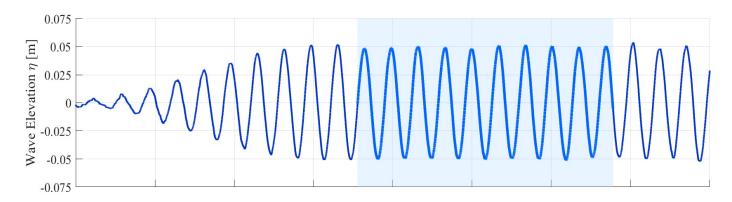
Wave stability criteria:

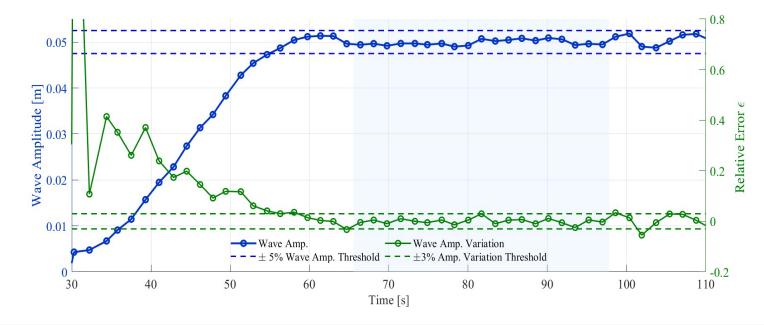
- <5% variation from the set amplitude
- <3% cycle to cycle amplitude fluctuation

Torque stability criteria:

<7% cycle to cycle torque fluctuation

Selected cycles are combined with those from repeated tests conducted under similar conditions.





















Selection of Wave Conditions

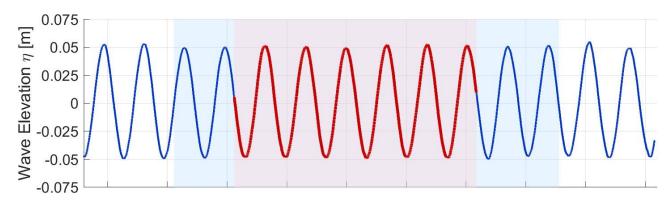
Wave stability criteria:

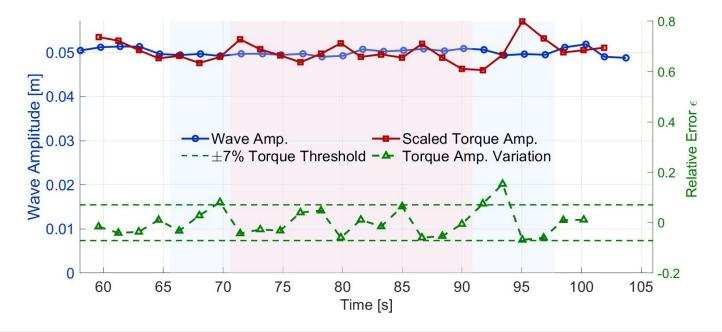
- <5% variation from the set amplitude
- <3% cycle to cycle amplitude fluctuation

Torque stability criteria:

<7% cycle to cycle torque fluctuation

Selected cycles are combined with those from repeated tests conducted under similar conditions.















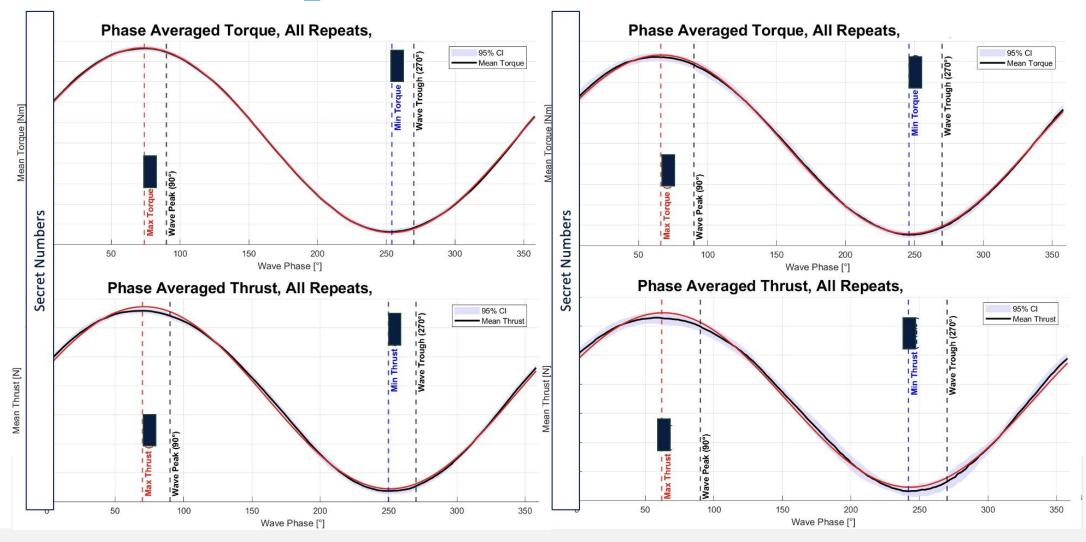








Phase-averaged results











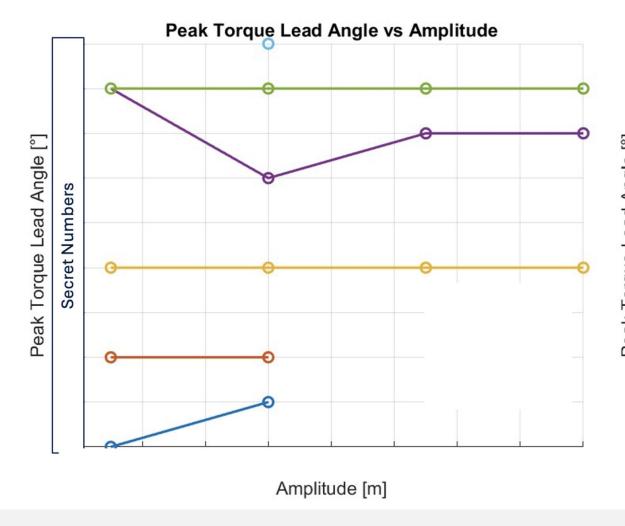


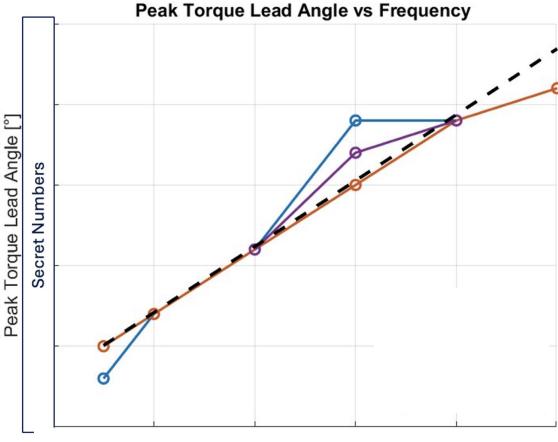






Torque phase lag





Wave Set Frequency [Hz]













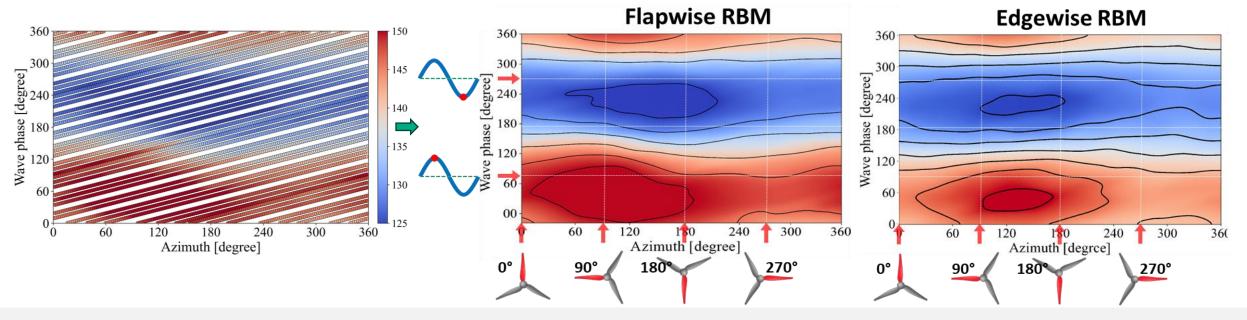






Unsteady loading in waves

- RBMs from all 3 blades selected over stable wave cycles across all repeated tests,
- RBM data visualized in both wave phase and blade phase (azimuthal) coordinates,
- Flapwise and edgewise load maxima / minima do not occur in phase with wave crest / trough and blade top / bottom dead centre positions,
- **Hypothesis**: wave-induced perturbations correlated loads well along blade spans when blades near horizontal, but decorrelate loads when blades vertical due to depth decay.















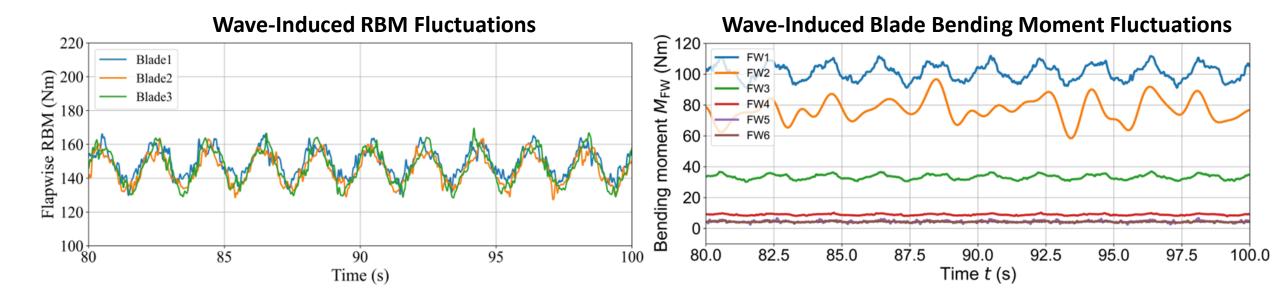






Unsteady loading in waves

- RBM and bending moments along span are analyzed to quantify unsteady blade loading
- Wave-induced unsteady load amplitudes can reach up to 30% of the steady-state blade load (at H/D=0.0625)

















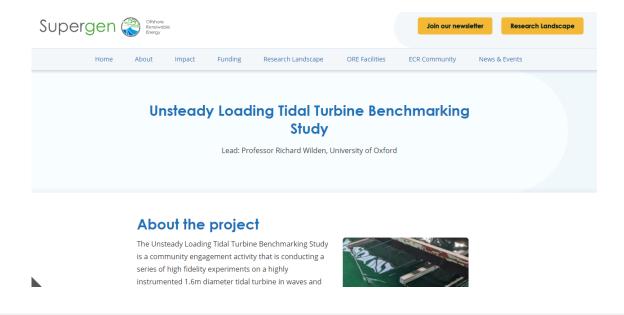


How to Participate?

For further details on the Tidal Turbine Benchmarking Project, including benchmark data and how to take part:

https://supergen-ore.net/projects/tidal-turbine-benchmarking Email Richard Willden Richard.Willden@eng.ox.ac.uk or Xiaosheng Chen xiaosheng.chen@eng.ox.ac.uk

- 1. Download **geometry data and test conditions** from the repository links on the Supergen website.
- 2. Perform blind predictions.
- 3. Download example data **submission file** and submission data formatting guide from the repository links on the Supergen website.
- 4. Upload data in specified format to us.



















Data Depository & Test Conditions

Turbine geometry:

- 3D CAD geometry of nacelle and tower
- 2D hydrofoil sections / chord and twist distributions
- 2D hydrofoil CFD data and link to experimental data
- 3D CAD geometry of blade
- Turbulence grid geometry:
 - 3D CAD data
- Test conditions:
 - TSR range / flow velocities
 - Measured wave heights and frequencies



















Benchmarking Test Cases

- The table below illustrates all the wave conditions tested during the March 2025 campaign, all cases are tested under a tow-speed of $1.0 \ m/s$ and a rotation RPM of $72.0 \ (TSR \approx 6.03)$
- Depending on the modelling methodology simulation of more or less cases may be possible
- The **blue** cases are the cases with best quality data, and those with the "**Priority**" tag are the ones **requested to be attempted by all simulation methodologies**. Other cases are optional and welcomed.
- 7 solid wave gauges and 6 ultrasonic probes mounted on the carriage to measure wave elevation. U1 signals are used as the reference wave elevation.
 Wave Test Matrix

 Wave Gauge Locations

Wave Gauge Locations Wave Paddle Frequency [Hz] •G5 4 • G4 0.225 0.25 0.3 0.35 0.45 0.5 0.4 • G3 U5 0.025 Priority • G2 y (m) Wave 0.035 Ú3 **Turbine Amplit** 0.05 **Priority Priority Priority** Priority Priority ude [m] -2U40.075 -3× Ultrasonic Probe Wave Gauge 0.1 Priority Priority **Green: More limited data available** Blue: Best quality data x(m)









Priority: overlap cases from both 2022 and 2025 campaigns - previously specified for benchmarking





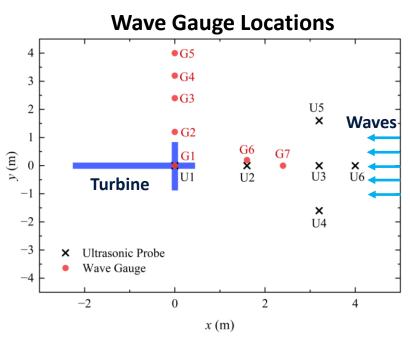




Benchmarking Test Cases

- The previously mentioned wave case frequencies are the wave paddle frequencies.
- A table is provided below to translate between the wave paddle frequency and the rotor encounter wave frequency using the wave dispersion equation.

Wave Paddle Frequency f_0 [Hz]	0.225	0.250	0.300	0.350	0.400	0.450	0.500
Wave Number k	0.238	0.278	0.375	0.498	0.646	0.816	1.007
Wave Celerity c_0 [m/s]	5.945	5.649	5.023	4.415	3.892	3.465	3.119
Wave Encounter Velocity c_0 + Uinf [m/s]	6.945	6.649	6.023	5.415	4.892	4.465	4.119
Wave Encounter Frequency f_en [Hz]	0.263	0.294	0.360	0.429	0.503	0.580	0.660
Wavelength L [m]	26.424	22.595	16.743	12.615	9.730	7.700	6.239
h/L	1.284	1.502	2.027	2.689	3.487	4.406	5.439









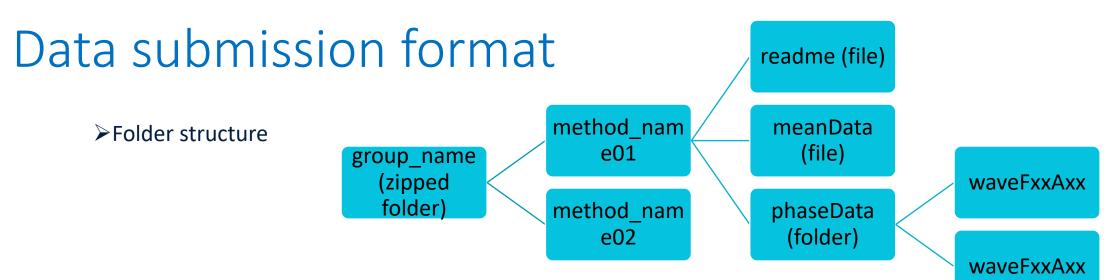












Please be noted that the numbers in the waveFxxAxx filename needs to be exactly 3 decimal places.

			1	Wave Paddle F	requency [Hz			
Wave Amplitude [m]		0.225	0.250	0.300	0.350	0.400	0.450	0.500
	0.025			waveF0.300A 0.025				
	0.035							
	0.050	waveF0.225A 0.050	waveF0.250A 0.050	waveF0.300A 0.050		waveF0.400A 0.050		waveF0.500A 0.050
	0.075							
	0.100			waveF0.300A 0.100		waveF0.400A 0.100		

















Data submission format

- Format of the **readme** file: include necessary detail of the simulation methodology, sub-models, assumptions, conditions, domain size etc.
- > Format of the meanData file:
 - Include time-averaged rotor integrated quantities from all simulated wave cases.
 - File format: **tab-delimited ASCII file**, with header line of parameter names start with "#", and each column separated by a tab, numbers should be rounded to **6 decimal places**.
 - Oct: thrust coefficient, Cp: power coefficient, RBM: root bending moment, FW: flapwise, EW: edgewise
 - o mean: time-averaged data, SE: standard error $\sigma_{\overline{C_T}} \approx \frac{\sigma_{C_T}}{\sqrt{n}}$, STD: standard deviation

#wave Amplit ude [m]	waveFr equenc y [Hz]	Ct (mean)	Ct (SE)	Ct (STD)	Cp (mean)	Cp (SE)	Cp (STD)	RBM_FW (mean) [Nm]	RBM_FW (SE) [Nm]	RBM_FW (STD) [Nm]	RBM_EW (mean) [Nm]	RBM_EW (SE) [Nm]	RBM_EW (STD) [Nm]
0.0250	0.2250	xxx	xxx	xxx	xxx	XXX	xxx	xxx	xxx	xxx	xxx	xxx	xxx
0.0250 00	0.2500 00	xxx	xxx	xxx	xxx	XXX	xxx	xxx	xxx	xxx	xxx	xxx	xxx
etc.													











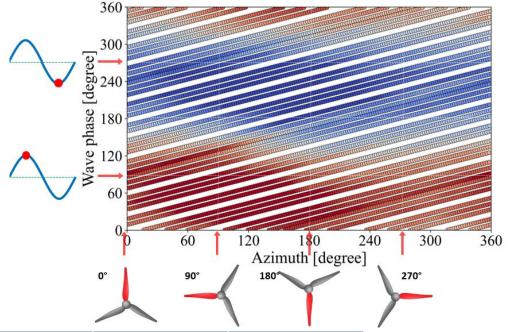






Data submission format

- Format of the **phaseData** file
 - o Include time- and phase-history data (Ct, Cp, RBMs) of a single blade for each extracted/sampled timestep with calculated wave and rotational phase angles (examples on the right).
 - Note: history of all 3 blades can be included with a shifted rotational phase to increase the data size.
 - File format: tab-delimited ASCII file, with header line of parameter names start with "#", and each column separated by a tab, numbers should be rounded to up to 6 decimal places.
 - Arranged as below:



#physicalTime [s]	wavePhase [deg]	bladePhase [deg]	Ct	Ср	RBM_FW [Nm]	RBM_EW [Nm]
0.000001	0.00000	xxx	xxx	xxx	xxx	xxx
0.000002	3.00000	xxx	xxx	xxx	xxx	xxx
etc.						











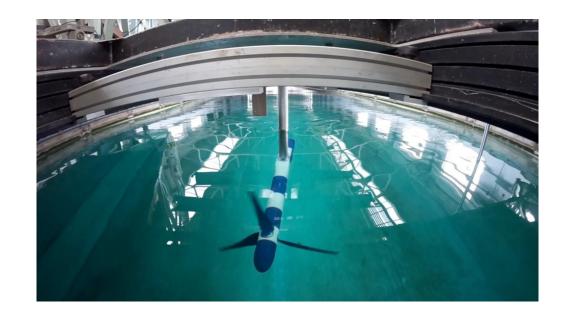






Advice for Modellers

- The exercise is not a competition but aims to improve the understanding of the relative strengths and weaknesses and limitations of the different modelling approaches,
- Experiments are also **imperfect** so we do not expect any simulation data to perfectly match the measurements.



Benchmarking Timeline

- Register your participation in the Stage II: Unsteady Loading in Waves benchmarking exercise by email to Xiaosheng Chen xiaosheng.chen@eng.ox.ac.uk
- 31st October 2025 Webinar with registered participants to clarify case set up and data submission requirements.
- Nov / Dec 2025 (TBC) optional progress Webinar
- Submit your blind prediction loading solutions by 16th January 2026.

















Participation & benchmarking data

For further details on the Tidal Turbine Benchmarking Project, including benchmark data and how to take part:

https://supergen-ore.net/projects/tidal-turbine-benchmarking Email Richard Willden <u>Richard.Willden@eng.ox.ac.uk</u> or Xiaosheng Chen <u>xiaosheng.chen@eng.ox.ac.uk</u>



Stage I – Uniform Flow benchmarking data

Data repository currently being uploaded to the website (in the "Released Data Log")

Experimental data: Tucker Harvey et al. "Tidal Turbine Benchmarking Project: Stage I – Steady Flow Experiments" Blind predictions: Willden et al. "Tidal Turbine Benchmarking Project: Stage I – Steady Flow Blind Predictions" Full comparisons in companion Journal Articles in submission to Journal of Fluids & Structures

Stage II – Unsteady Loading benchmarking data

Data to be made available following final submissions to the blind prediction exercise on 16th January 2026 Wave loading data analysis ... in preparation

Yawed loading data analysis ... in preparation















